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- Review: prototype 2nd-order system
- Today's topic: transient response specifications


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Reading: FPE, Sections 3.3-3.4; lab manual

## Prototype 2nd-Order System

$$
H(s)=\frac{\omega_{n}^{2}}{s^{2}+2 \zeta \omega_{n} s+\omega_{n}^{2}}
$$

By the quadratic formula, the poles are:

$$
\begin{aligned}
s & =-\zeta \omega_{n} \pm \omega_{n} \sqrt{\zeta^{2}-1} \\
& =-\omega_{n}\left(\zeta \pm \sqrt{\zeta^{2}-1}\right)
\end{aligned}
$$

The nature of the poles changes depending on $\zeta$ :

- $\zeta>1$ both poles are real and negative
- $\zeta=1 \quad$ one negative pole
- $\zeta<1$ two complex poles with negative real parts

$$
s=-\sigma \pm j \omega_{d}
$$

where

$$
\sigma=\zeta \omega_{n}, \omega_{d}=\omega_{n} \sqrt{1-\zeta^{2}}
$$

## Prototype 2nd-Order System

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H(s)=\frac{\omega_{n}^{2}}{s^{2}+2 \zeta \omega_{n} s+\omega_{n}^{2}}, \quad \zeta<1
$$

The poles are

$$
s=-\zeta \omega_{n} \pm j \omega_{n} \sqrt{1-\zeta^{2}}=-\sigma \pm j \omega_{d}
$$



Note that

$$
\begin{aligned}
\sigma^{2}+\omega_{d}^{2} & =\zeta^{2} \omega_{n}^{2}+\omega_{n}^{2}-\zeta^{2} \omega_{n}^{2} \\
& =\omega_{n}^{2} \\
\cos \varphi & =\frac{\zeta \omega_{n}}{\omega_{n}}=\zeta
\end{aligned}
$$

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Let's compute the system's impulse and step response:

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where $\sigma=\zeta \omega_{n}$ and $\omega_{d}=\omega_{n} \sqrt{1-\zeta^{2}}$ (damped frequency)

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The parameter $\zeta$ is called the damping ratio

- $\zeta>1$ : system is overdamped
- $\zeta<1$ : system is underdamped
- $\zeta=0$ : no damping $\left(\omega_{d}=\omega_{n}\right)$


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We will see that the parameters $\zeta$ and $\omega_{n}$ determine certain important features of the transient part of the above step response.

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We will see that the parameters $\zeta$ and $\omega_{n}$ determine certain important features of the transient part of the above step response.

We will also learn how to pick $\zeta$ and $\omega_{n}$ in order to shape these features according to given specifications.

## Transient Response Specifications: Rise Time

Let's first take a look at 1st-order step response

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Examples of rise time:

- car - going from 0 to 60 mph in 7 sec
- oven - reach desired preheat temperature quickly
- thermostat, building climate control
- other examples?


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& t_{r}=t_{0.9}-t_{0.1}=\frac{\ln 0.9-\ln 0.1}{a}=\frac{\ln 9}{a} \approx \frac{2.2}{a}
\end{aligned}
$$

## Transient Response Specs

Now let's consider the more interesting case: $2 n d$-order response

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- overshoot $M_{p}$ and peak time $t_{p}$
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Trade-offs among specs:

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Trade-offs among specs: decrease $t_{r} \longrightarrow$ increase $M_{p}$, etc.


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So, we will work with $t_{r} \approx \frac{1.8}{\omega_{n}} \quad(\operatorname{good}$ approx. when $\zeta \approx 0.5)$

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so $t_{p}=\frac{\pi}{\omega_{d}}$

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We have just computed $t_{p}=\frac{\pi}{\omega_{d}}$

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M_{p}=y\left(t_{p}\right)-1
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## Formulas for TD Specs: Overshoot \& Peak Time



We have just computed $t_{p}=\frac{\pi}{\omega_{d}}$
To find $M_{p}$, plug this value into $y(t)$ :

$$
M_{p}=y\left(t_{p}\right)-1=-e^{-\frac{\sigma \pi}{\omega_{d}}}\left(\cos \left(\omega_{d} \frac{\pi}{\omega_{d}}\right)+\frac{\sigma}{\omega_{d}} \sin \left(\omega_{d} \frac{\pi}{\omega_{d}}\right)\right)
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& =\exp \left(-\frac{\sigma \pi}{\omega_{d}}\right)=\exp \left(-\frac{\pi \zeta}{\sqrt{1-\zeta^{2}}}\right) \quad \text { - exact formula }
\end{aligned}
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Formulas for TD Specs: Settling Time


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& t_{s}=\min \left\{t>0: \frac{\left|y\left(t^{\prime}\right)-y(\infty)\right|}{y(\infty)} \leq 0.05 \text { for all } t^{\prime} \geq t\right\} \text { (here, } \\
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$\pm 1)$, so $e^{-\sigma t_{s}} \leq 0.05$
this gives $t_{s}=-\frac{\ln 0.05}{\sigma} \approx \frac{3}{\sigma}$

Formulas for TD Specs

$$
H(s)=\frac{\omega_{n}^{2}}{s^{2}+2 \zeta \omega_{n} s+\omega_{n}^{2}}=\frac{\sigma^{2}+\omega_{d}^{2}}{(s+\sigma)^{2}+\omega_{d}^{2}}
$$

$$
\begin{aligned}
t_{r} & \approx \frac{1.8}{\omega_{n}} \\
t_{p} & =\frac{\pi}{\omega_{d}} \\
M_{p} & =\exp \left(-\frac{\pi \zeta}{\sqrt{1-\zeta^{2}}}\right) \\
t_{s} & \approx \frac{3}{\sigma}
\end{aligned}
$$

## TD Specs in Frequency Domain

We want to visualize time-domain specs in terms of admissible pole locations for the 2nd-order system

$$
H(s)=\frac{\omega_{n}^{2}}{s^{2}+2 \zeta \omega_{n} s+\omega_{n}^{2}}=\frac{\sigma^{2}+\omega_{d}^{2}}{(s+\sigma)^{2}+\omega_{d}^{2}}
$$

where $\sigma=\zeta \omega_{n}$

$$
\omega_{d}=\omega_{n} \sqrt{1-\zeta^{2}}
$$

Step response: $y(t)=1-e^{-\sigma t}\left(\cos \left(\omega_{d} t\right)+\frac{\sigma}{\omega_{d}} \sin \left(\omega_{d} t\right)\right)$


$$
\begin{aligned}
\omega_{n}^{2} & =\sigma^{2}+\omega_{d}^{2} \\
\zeta & =\cos \varphi
\end{aligned}
$$

## Rise Time in Frequency Domain

Suppose we want $t_{r} \leq c \quad$ ( $c$ is some desired given value)

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## Rise Time in Frequency Domain

Suppose we want $t_{r} \leq c \quad(c$ is some desired given value)
$t_{r} \approx \frac{1.8}{\omega_{n}} \leq c \quad \Longrightarrow \quad \omega_{n} \geq \frac{1.8}{c}$
Geometrically, we want poles to lie in the shaded region:

(recall that $\omega_{n}$ is the magnitude of the poles)

## Overshoot in Frequency Domain

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\end{aligned} \\
& \text { - need } \varphi \text { to be small } \\
& \text { Intuition: good damping } \rightarrow \\
& \text { good decay in } 1 / 2 \text { period }
\end{aligned}
$$

## Settling Time in Frequency Domain

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## Settling Time in Frequency Domain

Suppose we want $t_{s} \leq c$
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Want poles to be sufficiently fast (large enough magnitude of real part):


Intuition: poles far to the left $\rightarrow$ transients decay faster $\rightarrow$ smaller $t_{s}$

## Combination of Specs

If we have specs for any combination of $t_{r}, M_{p}, t_{s}$, we can easily relate them to allowed pole locations:


The shape and size of the region for admissible pole locations will change depending on which specs are more severely constrained.

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But: not very rigorous, and also only valid for our prototype 2nd-order system, which has only 2 poles and no zeros ...

