ECE 486: Control Systems

#### ► Lecture 17C: lead/lag control

#### Goal: introduce the use of lead and lag dynamic compensators

Reading: FPE, Chapter 5

## Approximate PD Using Dynamic Compensation

Reminder: we can approximate the D-controller  $K_{\rm D}s$  by

$$K_{\rm D} \frac{ps}{s+p} \longrightarrow K_{\rm D}s \text{ as } p \to \infty$$

— here, -p is the *pole* of the controller.

So, we replace the PD controller  $K_{\rm P} + K_{\rm D}s$  by

$$K(s) = K_{\rm P} + K_{\rm D} \frac{ps}{s+p}$$



## Lead & Lag Compensators

Consider a general controller of the form

$$K \frac{s+z}{s+p}$$
 —  $K, z, p > 0$  are design parameters

Depending on the relative values of z and p, we call it:

- ▶ a lead compensator when z < p
- a lag compensator when z > p

Why the name "lead/lag?" — think frequency response

$$\angle \frac{j\omega + z}{j\omega + p} = \angle (j\omega + z) - \angle (j\omega + p) = \psi - \phi$$

- if z < p, then  $\psi \phi > 0$  (phase lead)
- if z > p, then  $\psi \phi < 0$  (phase lag)



# Summary on Design Trade-offs

Some deign trade-offs for the lead control:

- p large good damping, but bad noise suppression (too close to PD)
- ▶ *p* small noise suppression is better, but worse tracking performance
- ▶ intermediate values of p how to set the control gains?

We will use the Bode plot to do the design.

## Lead Compensation: Bode Plot

$$KD(s) = K\frac{s+z}{s+p}, \qquad p \gg z$$

In Bode form:

$$KD(s) = \frac{Kz\left(\frac{s}{z}+1\right)}{p\left(\frac{s}{p}+1\right)}$$

or, absorbing z/p into the overall gain, we have

$$KD(s) = \frac{K\left(\frac{s}{z}+1\right)}{\left(\frac{s}{p}+1\right)}$$

#### Break-points:

- ▶ Type 1 zero with break-point at  $\omega = z$  (comes first,  $z \ll p$ )
- ▶ Type 1 pole with break-point at  $\omega = p$

# Lead Compensation: Bode Plot

$$KD(s) = \frac{K\left(\frac{s}{z}+1\right)}{\left(\frac{s}{p}+1\right)}$$



 magnitude levels off at high frequencies => better noise suppression

adds phase, hence the term "phase lead"

# Lead Compensation and Phase Margin

$$KD(s) = \frac{K\left(\frac{s}{z}+1\right)}{\left(\frac{s}{p}+1\right)}$$



For best effect on PM,  $\omega_c$ should be halfway between zand p (on log scale):

$$\log \omega_c = \frac{\log z + \log p}{2}$$
  
or  $\omega_c = \sqrt{z \cdot p}$ 

— geometric mean of z and p

Trade-offs: large p - z means

- ▶ large PM (closer to  $90^{\circ}$ )
- but also bigger M at higher frequencies (worse noise suppression)

# Back to Our Example: $G(s) = \frac{1}{s^2}$

Objectives (same as before):

- stability
- good damping
- $\triangleright \omega_{\rm BW}$  close to 0.5

 $KG(s) = \frac{K}{s^2}$  (w/o lead):







— adding lead will increase  $\omega_c!!$ 

Back to Our Example:  $G(s) = \frac{1}{s^2}$ 



After adding lead with K = 1/4, what do we see?

▶ adding lead increases  $\omega_c$ 

$$\blacktriangleright \implies PM < 90^{\circ}$$

 $\blacktriangleright \implies \omega_{BW} \text{ may be } > \omega_c$ To be on the safe side, we choose a *new value* of K so that

$$\omega_c = \frac{\omega_{\rm BW}}{2}$$

(b/c generally  $\omega_c \leq \omega_{\rm BW} \leq 2\omega_c$ )

Thus, we want

$$\omega_c = 0.25 \implies K = \frac{1}{16}$$

Back to Our Example:  $G(s) = \frac{1}{s^2}$ 



Next, we pick z and p so that  $\omega_c$  is approximately their geometric mean:

e.g., 
$$z = 0.1, p = 2$$
  
 $\sqrt{z \cdot p} = \sqrt{0.2} \approx 0.447$ 

Resulting lead controller:

$$KD(s) = \frac{1}{16} \frac{\frac{s}{0.1} + 1}{\frac{s}{2} + 1}$$

(may still need to be refined using Matlab)

### Lead Controller Design Using Frequency Response General Procedure

- 1. Choose K to get desired bandwidth spec w/o lead
- 2. Choose lead zero and pole to get desired PM
  - in general, we should first check PM with the K from 1, w/o lead, to see how much more PM we need
- 3. Check design and iterate until specs are met.

This is an intuitive procedure, but it's not very precise, requires trial & error.